

# The Boundary Element Method in Elastostatics

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*4<sup>th</sup> April 2011*

*Introduction*

*Reciprocal  
theorem*

*Fundamental  
solutions*

*Boundary Integral  
Equation*

*Boundary Element  
Method*

*Re-analysis and  
Interactivity*

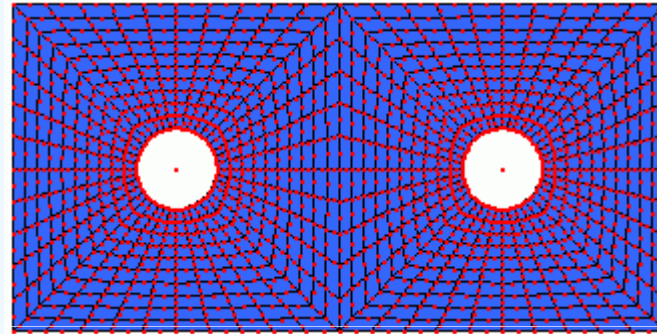
*Demonstration*

*Enrichment*

## Overview

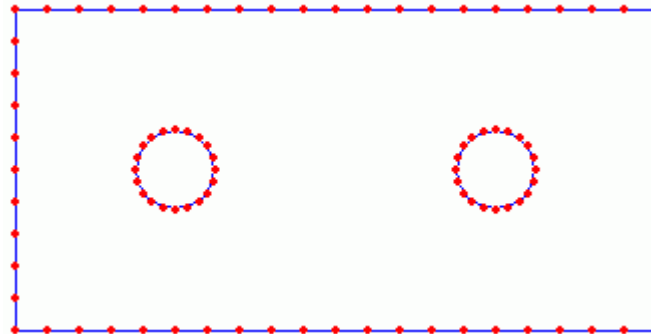
- Introduction
- Reciprocal theorem
- Fundamental solutions
- Boundary integral equation
- Boundary element method
- Re-analysis and interactivity
- Enrichment of approximation space

## Introduction – a cursory overview



### FEM

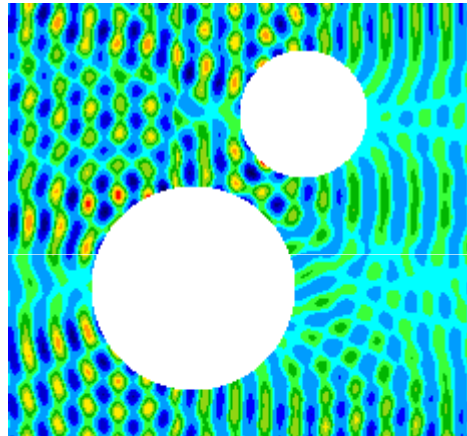
More versatile  
Domain method



### BEM

Computationally more  
demanding  
Simple meshing  
Solution accuracy

## Infinite domain problems



Elements are only used on the two internal boundaries in this wave problem

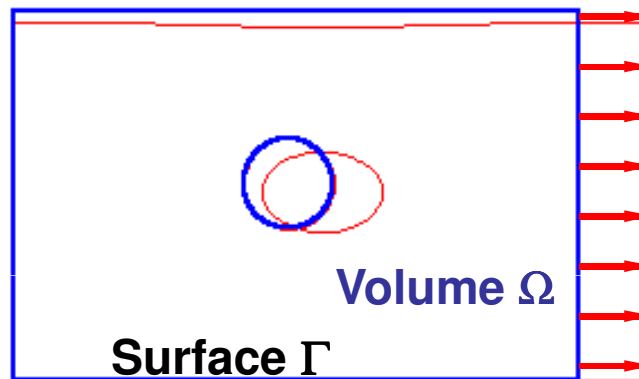
## Introduction – prelude to the theoretical development

For the purposes of this lecture we will start by stating the reciprocal theorem from a structural mechanics viewpoint.

The notes contain a fuller description with greater mathematical rigour. The reciprocal theorem is developed from a weighted residual expression.

We will confine ourselves to the collocation BEM. There is a popular Galerkin form of BEM too.

## Reciprocal theorem

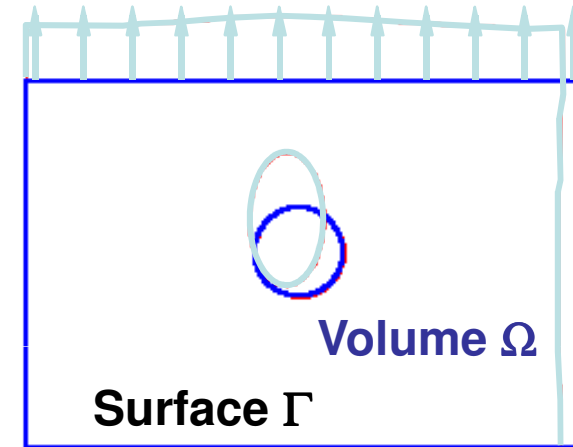


**Real load case:**

Tractions:  $t_i$

Displacements:  $u_i$

Body forces:  $b_i$



**Complementary load case:**

Tractions:  $t_i^*$

Displacements:  $u_i^*$

Body forces:  $b_i^*$

## Reciprocal theorem

Form statements of work done by **force** x **displacement**  
**force** x **displacement**

$$\int_{\Gamma} t_i^* u_i d\Gamma + \int_{\Omega} b_i^* u_i d\Omega = \int_{\Gamma} u_i^* t_i d\Gamma + \int_{\Omega} u_i^* b_i d\Omega$$

Integrate surface tractions over the surface  $\Gamma$

Integrate body forces over the volume  $\Omega$

This is the *reciprocal theorem* due to Maxwell and Betti.

## Fundamental solutions

Reciprocal theorem statement:

$$\int_{\Gamma} t_i^* u_i d\Gamma + \int_{\Omega} b_i^* u_i d\Omega = \int_{\Gamma} u_i^* t_i d\Gamma + \int_{\Omega} u_i^* b_i d\Omega$$

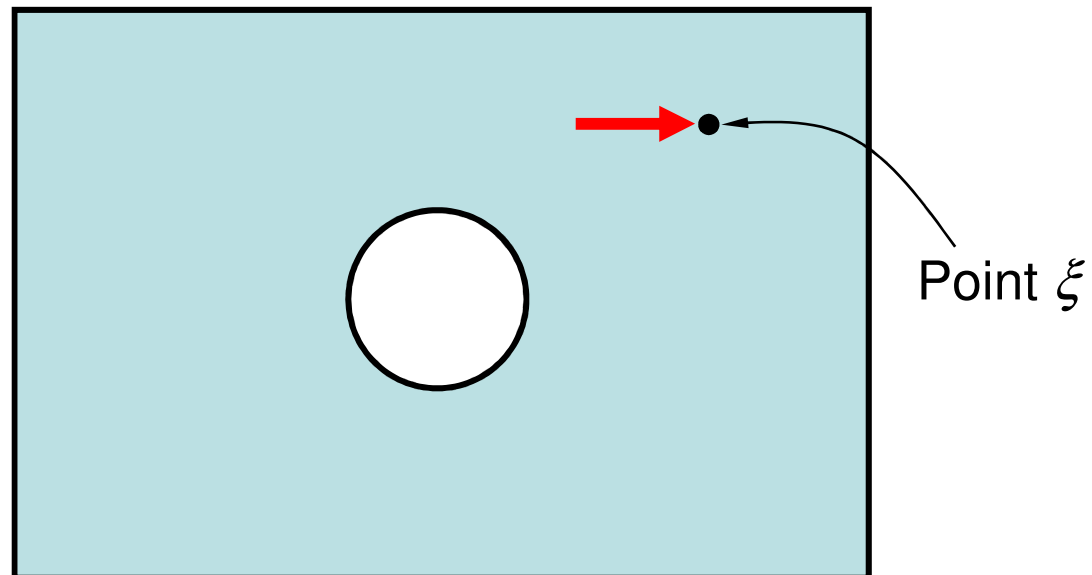
In order to reduce to boundary-only, we need to eliminate the two volume integrals.

- We will simplify the development for this lecture by stipulating no body forces in the real load case, i.e.  $b_i = 0$
- We will address the volume integral on the LHS by appropriate choice of a complementary load case



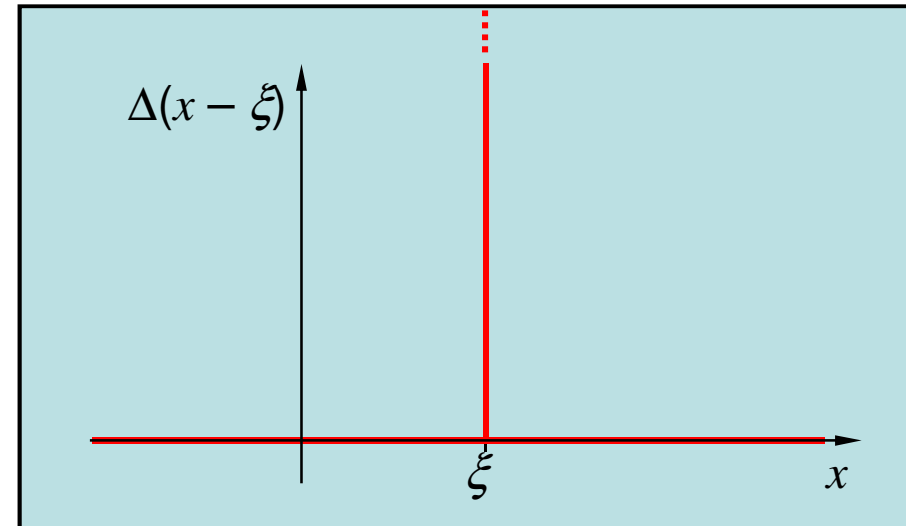
## Fundamental solutions

Complementary load case \*: Dirac delta function point load in one of the coordinate directions at some point  $\xi$



## Fundamental solutions

Properties of Dirac  
delta function:



$$\Delta(x - \xi) = \begin{cases} \infty & x = \xi \\ 0 & x \neq \xi \end{cases}$$

$$\int_a^b \Delta(x - \xi) dx = 1, \quad a < \xi < b$$

$$\int_a^b g(x) \Delta(x - \xi) dx = g(\xi), \quad a < \xi < b$$

## Fundamental solutions

So the displacement field in the complementary load case,  $u^*$ , is the solution to the equilibrium equation:

$$\sigma_{ij,j} + \Delta (x - \xi) e_i(x) = 0$$

## Fundamental solutions

So the displacement field in the complementary load case,  $u^*$ , is the solution to the equilibrium equation:

$$\sigma_{ij,j} + \Delta (x - \xi) e_i(x) = 0$$

It turns out that the solution is:

$$u_i^* = U_{ik} e_k$$

where

$$U_{ik} = \frac{1}{8\pi\mu(1-\nu)} \left[ (3-4\nu) \ln \left( \frac{1}{r} \right) \delta_{ik} + r_{,i} r_{,k} \right] \quad (2D)$$

$$U_{ik} = \frac{1}{16\pi\mu(1-\nu)r} [(3-4\nu)\delta_{ik} + r_{,i} r_{,k}] \quad (3D)$$

## Fundamental solution

So the displacement field  $u^*$ , is the solution to the e

$$\sigma_{ij,j} + \Delta$$

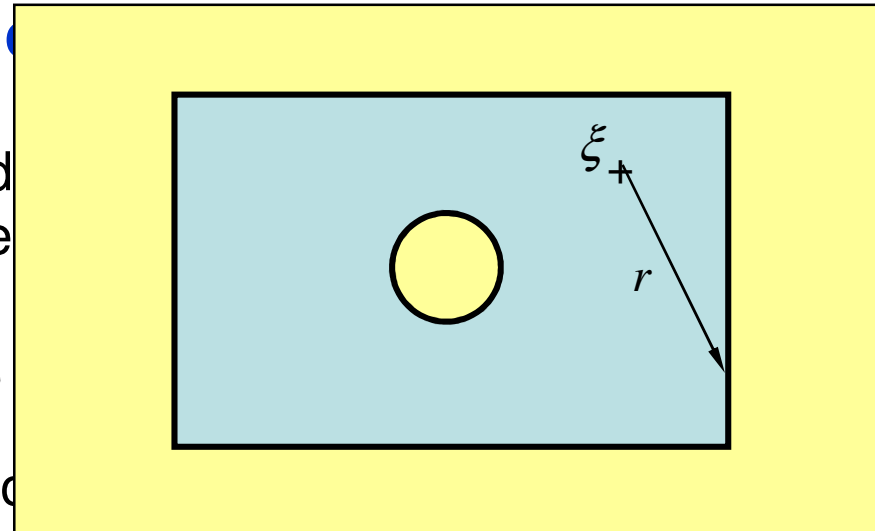
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$$U_{ik} = \frac{1}{16\pi\mu(1-\nu)r} [(3-4\nu)\delta_{ik} + r_{,i} r_{,k}] \quad (3D)$$



## Fundamental solutions

Displacement fundamental solutions:

$$U_{ik} = \frac{1}{8\pi\mu(1-\nu)} \left[ (3-4\nu) \ln\left(\frac{1}{r}\right) \delta_{ik} + r_{,i}r_{,k} \right] \quad (2D)$$

$$U_{ik} = \frac{1}{16\pi\mu(1-\nu)r} [(3-4\nu)\delta_{ik} + r_{,i}r_{,k}] \quad (3D)$$

Differentiate and apply Hooke's Law to arrive at the corresponding traction fundamental solutions:

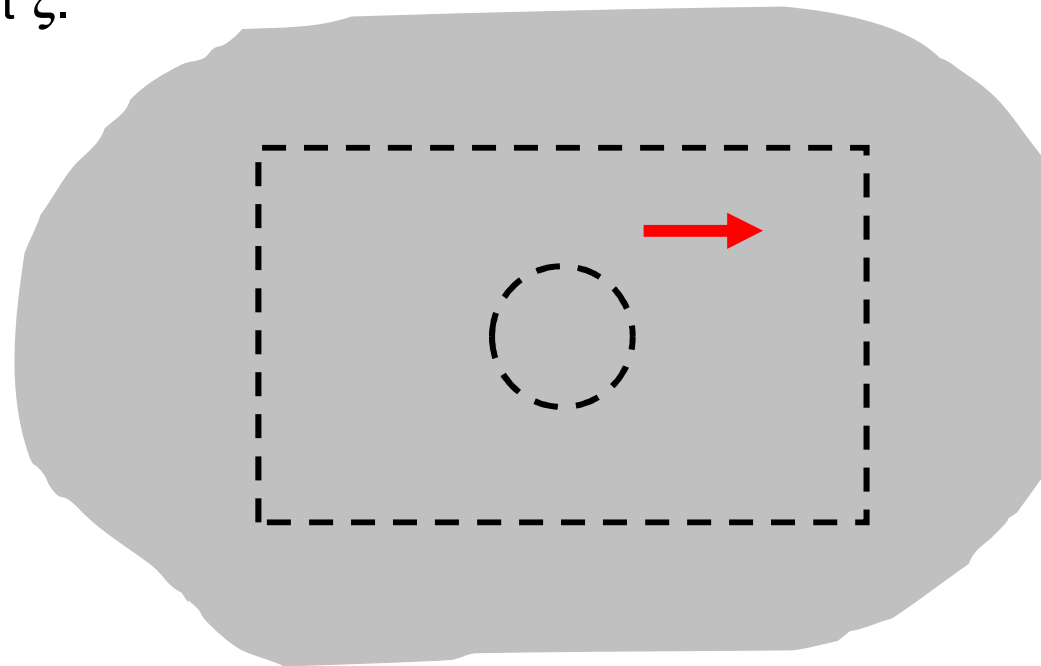
$$T_{ik} = \frac{-1}{4\pi(1-\nu)r} r_{,n} [(1-2\nu)\delta_{ik} + 2r_{,i}r_{,k}] + \frac{1-2\nu}{4\pi(1-\nu)r} (r_{,k}n_i - r_{,i}n_k) \quad (2D)$$

$$T_{ik} = \frac{-1}{8\pi(1-\nu)r^2} r_{,n} [(1-2\nu)\delta_{ik} + 3r_{,i}r_{,k}] + \frac{1-2\nu}{8\pi(1-\nu)r^2} (r_{,k}n_i - r_{,i}n_k) \quad (3D)$$

## Fundamental solutions

### *Physical significance:*

Fundamental solutions provide the displacement and traction fields, in an infinite material, due to a Dirac point force at  $\xi$ .



These solutions are due to Kelvin.

## Boundary integral equation

Use of the Dirac delta function as the complementary load case has given us a set of fundamental solutions from which we can easily find  $u^*$  and  $t^*$ .

The choice of the Dirac delta function also removes the remaining volume integral in the reciprocal theorem statement:

$$\int_{\Gamma} t_i^* u_i d\Gamma + \int_{\Omega} b_i^* u_i d\Omega = \int_{\Gamma} u_i^* t_i d\Gamma + \int_{\Omega} u_i^* b_i d\Omega$$

0 if  $b_i = 0$

$$\int_{\Omega} b_i^* u_i d\Omega = \int_{\Omega} \Delta(x - \xi) e_i u_i d\Omega = u_i(\xi) e_i$$



## Boundary integral equation

Treatment of the two volume integrals leaves simply:

$$u_k(\xi) + \int_{\Gamma} T_{ik} u_i d\Gamma = \int_{\Gamma} U_{ik} t_i d\Gamma \quad (*)$$

The last step in making this a boundary-only expression is to **move  $\xi$  to the boundary**, i.e.  $\xi \in \Gamma$ .

This causes complications because, when  $\xi \in \Gamma$ ,  $r$  passes through zero on the boundary causing both boundary integrals to **contain singular functions**.

## Boundary integral equation

The integrals containing  $U_{ik}$  are only weakly singular. The integrals in 2D, for example, have a logarithmic singularity and can be quickly evaluated using the logarithmic form of Gauss-Legendre quadrature.

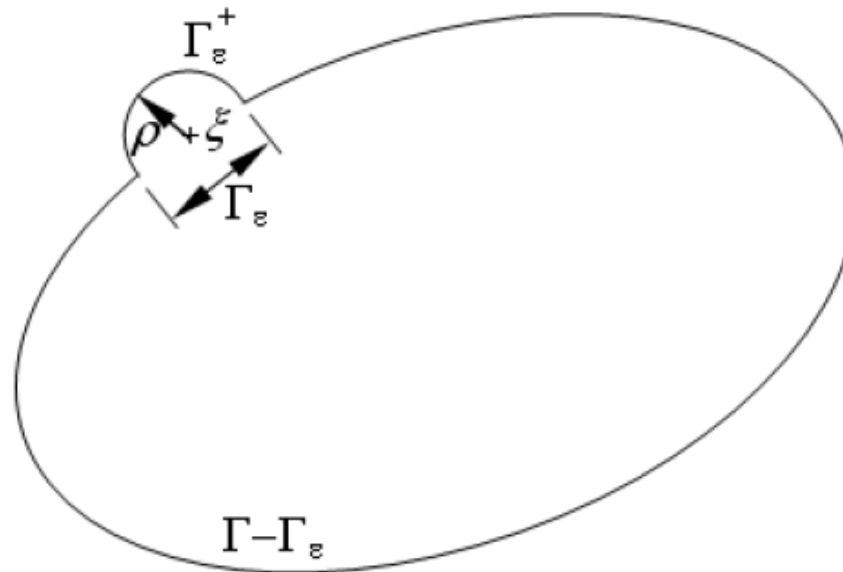
$$\int_{-1}^1 \ln \left( \frac{1}{x} \right) f(x) dx \simeq \sum_{i=1}^N f(x_i) w_i$$

There are other schemes – mostly involving coordinate transformation – for evaluating weakly singular integrals.

## Boundary integral equation

The strongly singular integrals containing  $T_{ik}$  may be taken in the *Cauchy principal value sense* (limit as radius  $\rho \rightarrow 0$ ) causing the introduction of a multiplier  $c_{ik}$ .

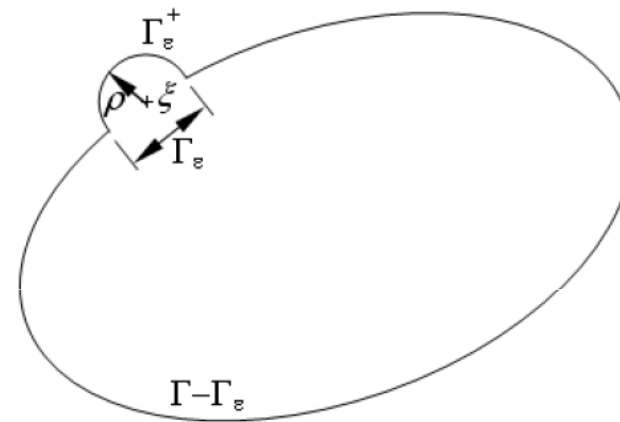
$$c_{ik}(\xi) u_k(\xi) + \int_{\Gamma} T_{ik} u_i d\Gamma = \int_{\Gamma} U_{ik} t_i d\Gamma$$



We will now see  
how this arises...

## Boundary integral equation

These integrals may be split into three parts:



$$\begin{aligned}
 \int_{\Gamma} T_{ik} u_i d\Gamma &= \lim_{\rho \rightarrow 0} \left\{ \int_{\Gamma - \Gamma_{\varepsilon}} T_{ik} u_i d\Gamma \right\} \\
 &+ \lim_{\rho \rightarrow 0} \left\{ \int_{\Gamma_{\varepsilon}^{+}} T_{ik} (u_i(x) - u_i(\xi)) d\Gamma(x) \right\} \rightarrow 0 \\
 &+ u_i(\xi) \lim_{\rho \rightarrow 0} \left\{ \int_{\Gamma_{\varepsilon}^{+}} T_{ik} d\Gamma \right\}
 \end{aligned}$$

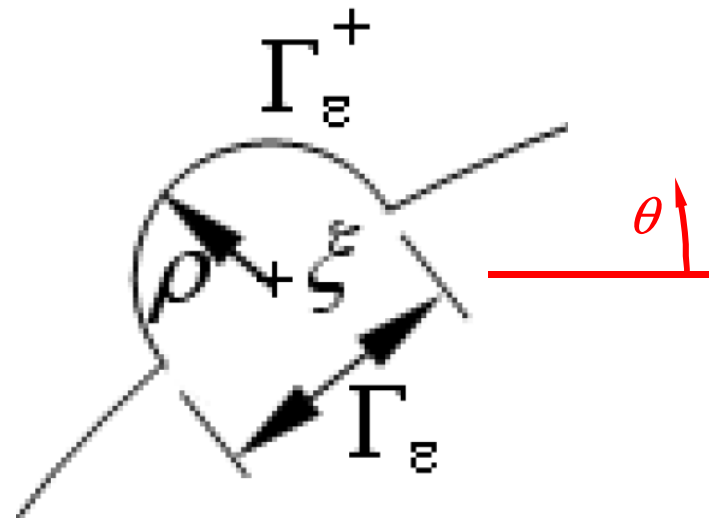
## Boundary integral equation

For convenience we write the last term as:

$$u_i(\xi) \lim_{\rho \rightarrow 0} \left\{ \int_{\Gamma_\varepsilon^+} T_{ik} d\Gamma \right\} = \alpha_{ik}(\xi) u_i(\xi)$$

So  $c_{ik} = \delta_{ik} + \alpha_{ik}$

Also define  $\theta$  coordinate:



## Boundary integral equation

On  $\Gamma_\varepsilon^+$ :

$$r = \rho \cos \theta e_1 + \rho \sin \theta e_2$$

$$d\Gamma_\varepsilon^+ = \rho d\theta$$

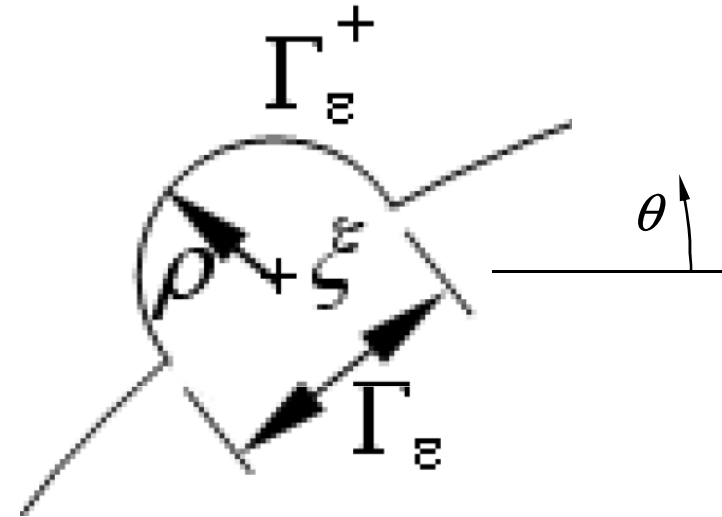
$$r_{,n} = 1$$

$$r_{,1} = \cos \theta$$

$$r_{,2} = \sin \theta$$

$$n_1 = \cos \theta$$

$$n_2 = \sin \theta$$



Substituting these into the various  $T_{ik}$  terms allows the integrals to be calculated analytically, yielding  $\alpha_{ik}$  and then  $c_{ik}$

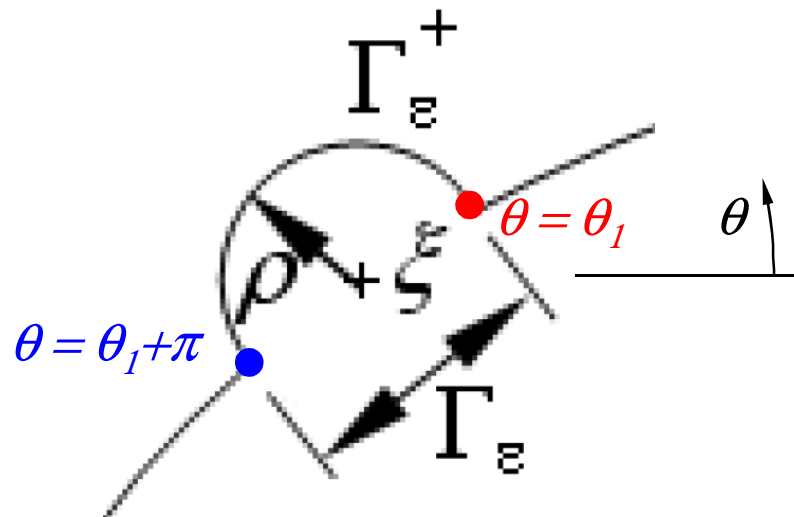
## Boundary integral equation

### Worked example: $\alpha_{11}$ on a smooth boundary

Substituting the functions of  $r$  and  $n$  into  $T_{11}$  gives

$$\alpha_{11}(\xi) = \int_{\Gamma_\varepsilon^+} \left[ \frac{-1}{4\pi(1-\nu)\rho} (1 - 2\nu + 2\cos^2\theta) \right] d\Gamma_\varepsilon^+$$

$$\alpha_{11}(\xi) = \frac{-1}{4\pi(1-\nu)} \int_{\theta_1}^{\theta_1+\pi} (1 - 2\nu + 2\cos^2\theta) d\theta$$



## Boundary integral equation

### Worked example: $\alpha_{11}$ on a smooth boundary

$$\alpha_{11}(\xi) = \frac{-1}{4\pi(1-\nu)} [(1-2\nu)\theta + (\theta + \cos\theta \sin\theta)]_{\theta_1}^{\theta_1+\pi}$$

$$\alpha_{11}(\xi) = \frac{-1}{4\pi(1-\nu)} \{(2-2\nu)\pi + \cos(\theta_1 + \pi) \sin(\theta_1 + \pi) - \cos\theta_1 \sin\theta_1\}$$

$$\alpha_{11}(\xi) = \frac{-1}{4\pi(1-\nu)} \{(2-2\nu)\pi + \sin\theta_1 \cos\theta_1 - \cos\theta_1 \sin\theta_1\}$$

$$\alpha_{11}(\xi) = -\frac{1}{2}$$

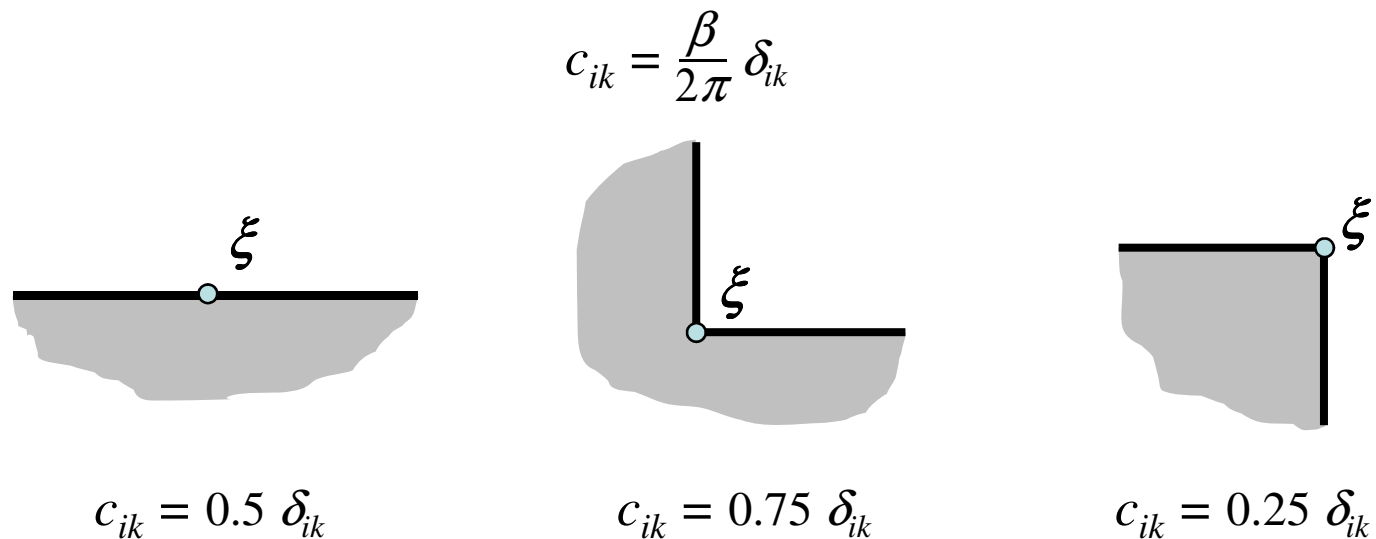
So

$$c_{11} = \delta_{11} + \alpha_{11} = 0.5$$



## Boundary integral equation

Generally the ‘free term’ or ‘jump term’  $c_{ik}(\xi)$  is determined by the angle  $\beta$  subtended by the material at  $\xi$ .



Still need to compute  $\lim_{\rho \rightarrow 0} \left\{ \int_{\Gamma - \Gamma_\epsilon} T_{ik} u_i d\Gamma \right\}$

***But.... much easier to use the row-sum method***

## Boundary integral equation

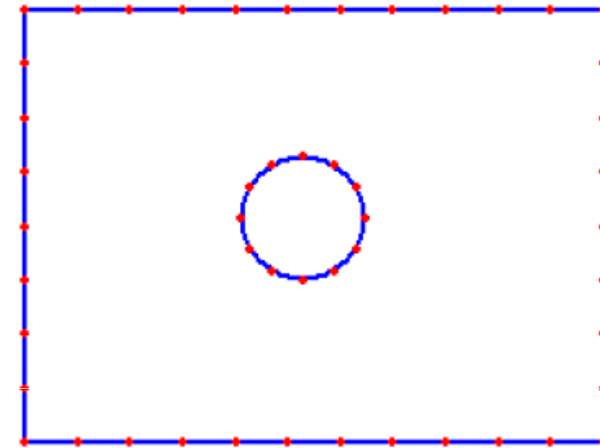
### Boundary integral equation

$$c_{ik}(\xi) u_k(\xi) + \int_{\Gamma} T_{ik} u_i d\Gamma = \int_{\Gamma} U_{ik} t_i d\Gamma$$

Analytical solution is possible only for the very simplest problems. We will proceed by discretisation, leading to the **Boundary Element Method** itself.

## Boundary element method

We discretise the boundary into elements.

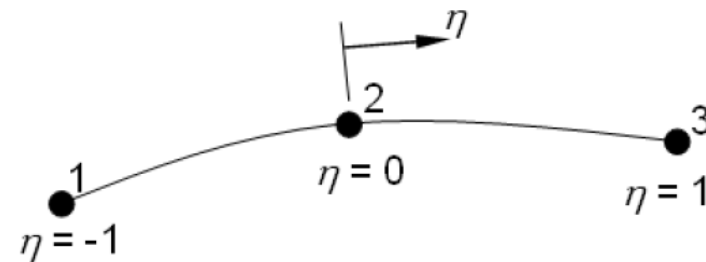


This provides:

- Nodal points and local interpolation to define an approximate solution, exactly like finite elements
- Convenient small portions of the boundary to perform numerical integration accurately
- A set of node points on which to place  $\xi$  in turn to provide a square system of linear equations

## Boundary element method

Boundary elements share many essential characteristics with finite elements:



$$N_1(\eta) = \frac{\eta}{2}(\eta - 1)$$

$$N_2(\eta) = (1 - \eta)(1 + \eta)$$

$$N_3(\eta) = \frac{\eta}{2}(\eta + 1)$$

$$u_i(\eta) = \sum_{j=1}^3 N_j(\eta) u_i^{jm}$$

## Boundary element method

Discrete form of the boundary integral equation:

$$c_{ik}(\xi) u_k(\xi) + \sum_{m=1}^M \int_{\Gamma_m} T_{ik} u_i d\Gamma = \sum_{m=1}^M \int_{\Gamma_m} U_{ik} t_i d\Gamma$$

Express  $u$  and  $t$  in their interpolated forms over element  $m$

$$c_{ik}(\xi) u_k(\xi) + \sum_{m=1}^M \int_{\Gamma_m} T_{ik} N_p u_i^{pm} d\Gamma = \sum_{m=1}^M \int_{\Gamma_m} U_{ik} N_p t_i^{pm} d\Gamma$$

Remove vectors of **nodal** displacements and tractions from the integrals:

$$c_{ik}(\xi) u_k(\xi) + \sum_{m=1}^M \int_{\Gamma_m} T_{ik} N_p d\Gamma u_i^{pm} = \sum_{m=1}^M \int_{\Gamma_m} U_{ik} N_p d\Gamma t_i^{pm}$$

## Boundary element method

$$c_{ik}(\xi) u_k(\xi) + \sum_{m=1}^M \int_{\Gamma_m} T_{ik} N_p d\Gamma u_i^{pm} = \sum_{m=1}^M \int_{\Gamma_m} U_{ik} N_p d\Gamma t_i^{pm}$$

Transform boundary integrals to local coordinate system:

$$c_{ik}(\xi) u_k(\xi) + \sum_{m=1}^M \int_{-1}^1 T_{ik} N_p J(\eta) d\eta u_i^{pm} = \sum_{m=1}^M \int_{-1}^1 U_{ik} N_p J(\eta) d\eta t_i^{pm}$$

Evaluate this equation for  $\xi$  at, say, node 1 and Dirac force in  $x$ -direction:

$$\begin{aligned} c_{11}(1) u_1^1 + \hat{h}_{1,1} u_1^1 + \hat{h}_{1,2} u_2^1 + \hat{h}_{1,3} u_1^2 + \hat{h}_{1,4} u_2^2 + \hat{h}_{1,5} u_1^3 + \dots \\ = g_{1,1} t_1^1 + g_{1,2} t_2^1 + g_{1,3} t_1^2 + g_{1,4} t_2^2 + g_{1,5} t_1^3 + \dots \end{aligned}$$

## Boundary element method

$$\begin{aligned} c_{11} (1) u_1^1 + \hat{h}_{1,1} u_1^1 + \hat{h}_{1,2} u_2^1 + \hat{h}_{1,3} u_1^2 + \hat{h}_{1,4} u_2^2 + \hat{h}_{1,5} u_1^3 + \dots \\ = g_{1,1} t_1^1 + g_{1,2} t_2^1 + g_{1,3} t_1^2 + g_{1,4} t_2^2 + g_{1,5} t_1^3 + \dots \end{aligned}$$

Embed the free term  $c$  into the others....

$$\begin{aligned} h_{1,1} u_1^1 + h_{1,2} u_2^1 + h_{1,3} u_1^2 + h_{1,4} u_2^2 + h_{1,5} u_1^3 + \dots \\ = g_{1,1} t_1^1 + g_{1,2} t_2^1 + g_{1,3} t_1^2 + g_{1,4} t_2^2 + g_{1,5} t_1^3 + \dots \end{aligned}$$

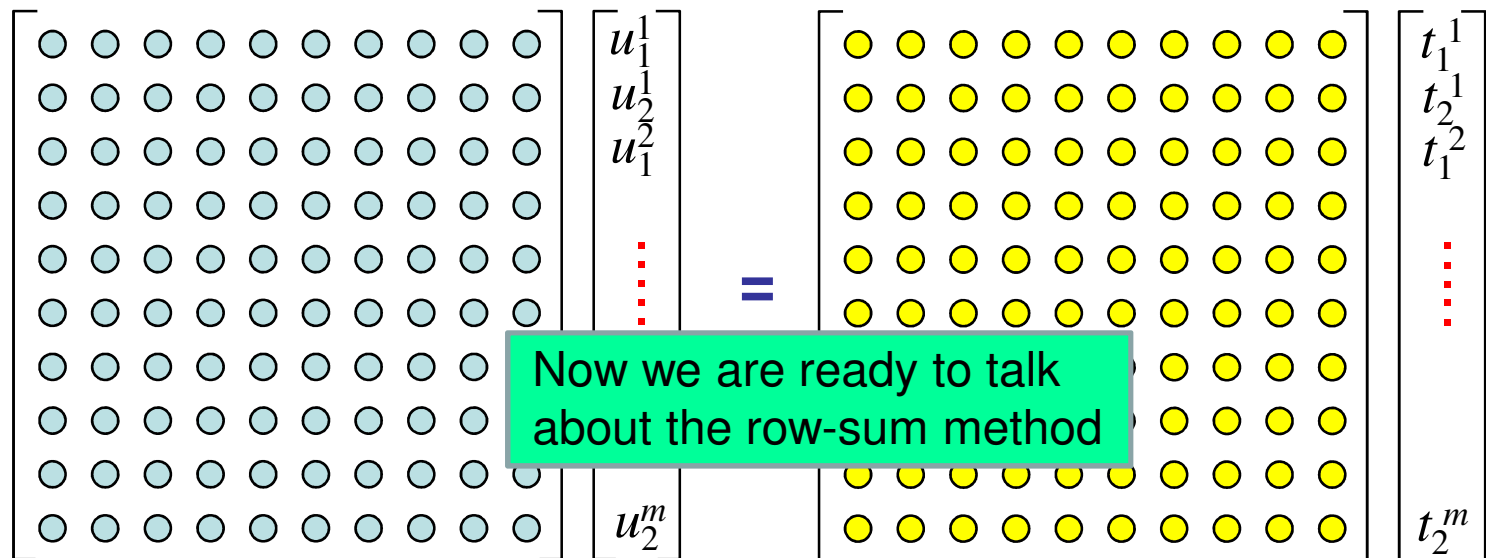
...by defining for notational simplicity

$$h_{i,j} = \hat{h}_{i,j} + \delta_{ij} c_{ij} (\xi)$$

## Boundary element method

Evaluating the  $h$  and  $g$  terms for  $\xi$  at each node in turn gives

$$H\underline{u} = G\underline{t}$$



Matrix of  $h$   
coefficients

Unknown displacements

Matrix of  $g$   
coefficients

Unknown tractions



## Boundary element method

For each row, prescribe either the displacement or traction as a boundary condition, and column-swap to bring all remaining unknowns to LHS...

$$\begin{bmatrix} \text{Matrix of } h \text{ and } g \text{ coefficients} \end{bmatrix} \begin{bmatrix} u_1^1 \\ t_2^1 \\ u_2^1 \\ \vdots \\ t_2^m \end{bmatrix} = \begin{bmatrix} \text{Matrix of } h \text{ and } g \text{ coefficients} \end{bmatrix} \begin{bmatrix} t_1^1 \\ u_2^1 \\ t_1^2 \\ \vdots \\ u_2^m \end{bmatrix}$$

Matrix of  $h$  and  $g$  coefficients

Unknown disp's & trac's

Matrix of  $h$  and  $g$  coefficients

Boundary conditions

## Boundary element method

The whole right hand side is now known and can be multiplied out leaving

$$A\underline{x} = \underline{b}$$

This can be solved either directly or iteratively.

Choice of solver is limited by asymmetry of  $A$ .

## Boundary element method

Now move  $\xi$  off the boundary and into the material. We can solve for the displacements at this internal point using the equation (\*) we developed part way through the derivation

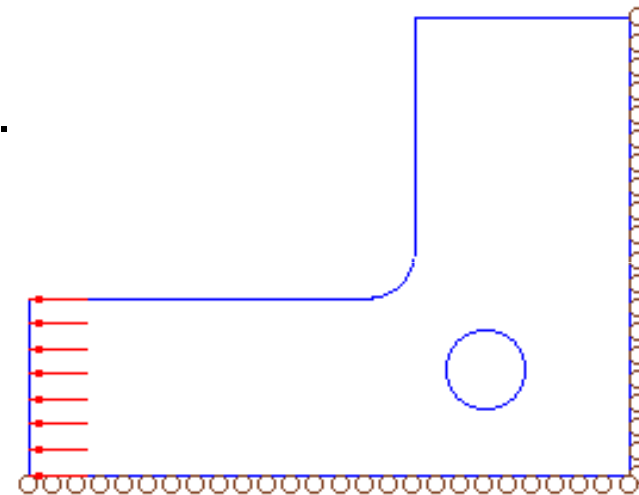
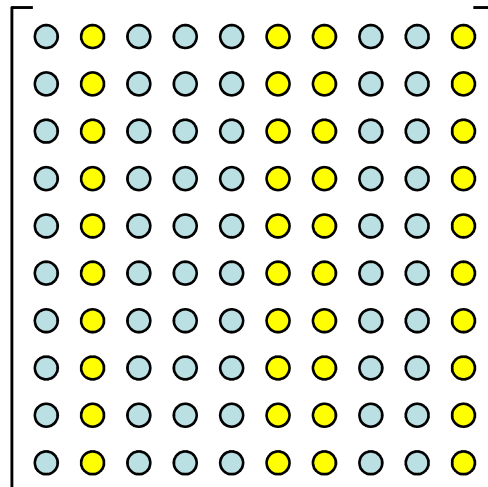
$$u_k(\xi) + \int_{\Gamma} T_{ik} u_i d\Gamma = \int_{\Gamma} U_{ik} t_i d\Gamma$$

Stress components at the internal point can be found from a derivative of this equation.

## Re-analysis and interactivity

One aspect of the BEM we are pursuing in Durham is re-analysis leading to an interactivity to design analysis.

A typical problem in elasticity....



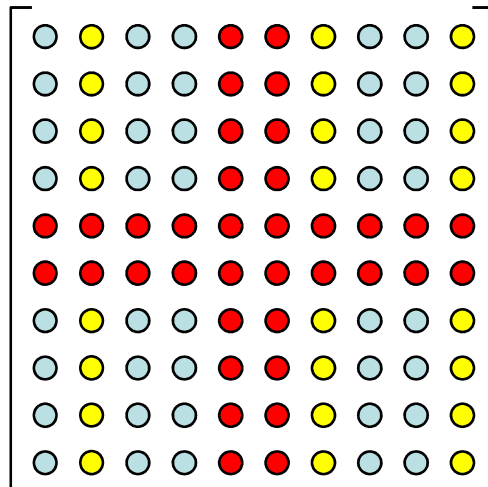
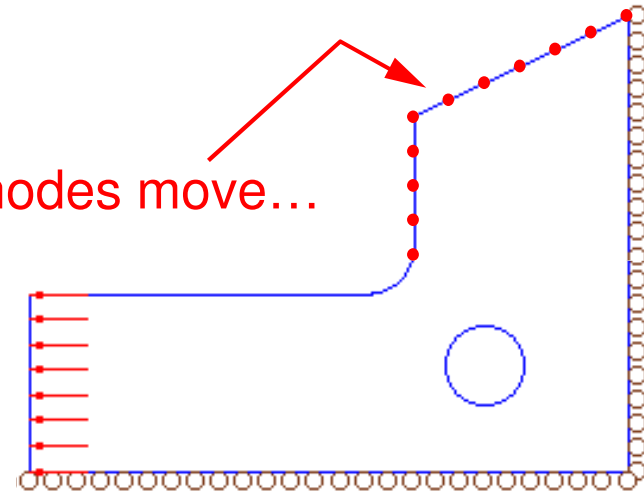
...and its matrix  $A$

## Re-analysis and interactivity

One aspect of the BEM we are pursuing here in Durham is re-analysis leading to an interactivity to design analysis.

Make a design change.....

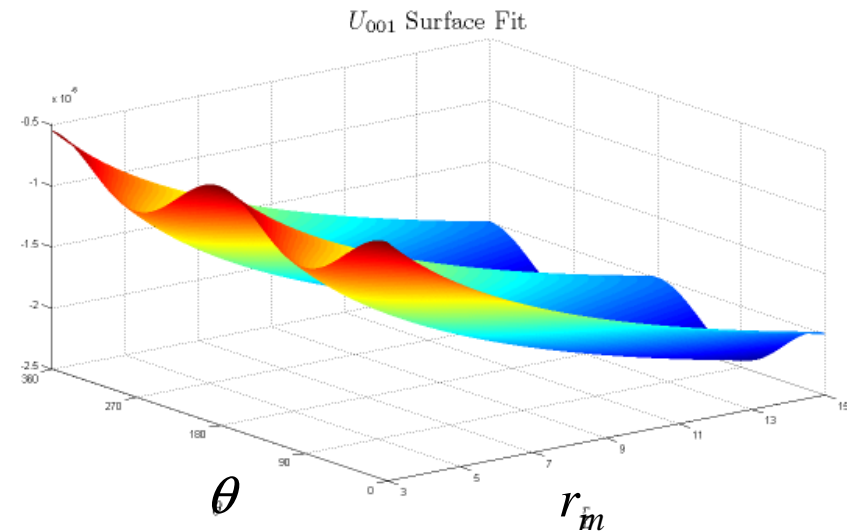
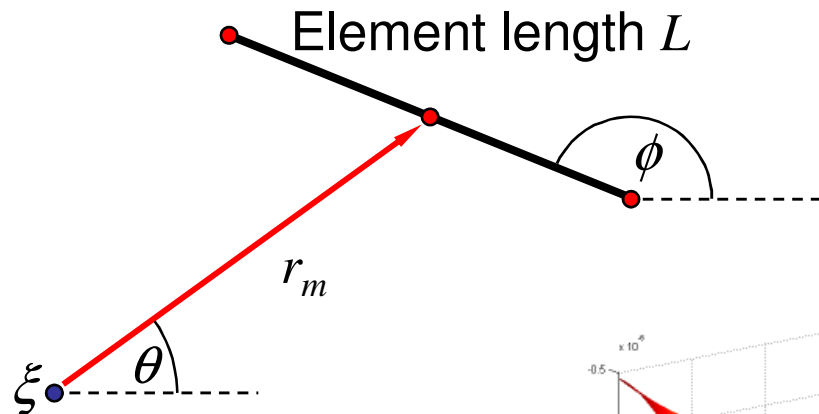
...only a few nodes move...



...and some rows and columns change, but most of the matrix is the same as the last one.

## Surface fits for rapid approximation of integrals

$$U_{001} = \left[ 2.166(1 + \cos(2\theta)) - 8.996 \ln\left(\frac{r_m}{L}\right) \right] \times 10^{-7}$$



## Iterative re-solution

We precondition a GMRES scheme with an approximate complete LU decomposition.

*First analysis:* LU-decomposition (save L and U)

*Re-analysis:* Use the full LU-decomposition of the **original** system as a preconditioner for iterative solution of the **perturbed** system.

| Preconditioner                      | Perturbation Type |             |                        |                          |                          |
|-------------------------------------|-------------------|-------------|------------------------|--------------------------|--------------------------|
|                                     | Move Point        | Move circle | External Fillet Resize | Internal Fillet Resize 1 | Internal Fillet Resize 2 |
| None                                | 30 – 50           | 31 – 49     | 36 – 48                | 34 – 50                  | 37 – 53                  |
| Diagonal                            | 39 – 53           | 36 – 47     | 36 – 44                | 43 – 47                  | 44 – 50                  |
| Full LU                             | 2 – 17            | 2 – 9       | 3 – 9                  | 3 – 13                   | 3 – 11                   |
| Number of iterations to convergence |                   |             |                        |                          |                          |

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# Demonstration



## Enriched approximation space

The shape functions form a Partition of Unity

$$\sum_{j=1}^3 N_j(\xi) = 1$$

We use this property to enrich using arbitrary functions

$$\sum_{j=1}^3 N_j(\xi) \psi(\xi) = \psi(\xi)$$

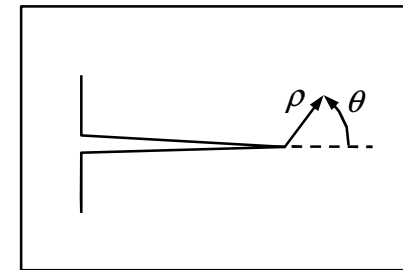
If we know functions  $\psi$  that populate the particular problem solution space we can include them in our approximation and obtain improved results .... Melenk & Babuška.

## Enriched dual BEM for fracture mechanics

We can base an enrichment on the first order terms of the Williams expansion for displacement components around a crack tip

$$u_j^n(\xi) = \sum_{a=1}^M N_a(\xi) u_j^{na} + \sum_{a=1}^M \sum_{l=1}^L N_a(\xi) \psi_l^U(\xi) A_{jl}^{na}$$

$$\psi^U(\rho, \theta) = \left\{ \begin{array}{l} \sqrt{\rho} \cos\left(\frac{\theta}{2}\right), \sqrt{\rho} \sin\left(\frac{\theta}{2}\right), \\ \sqrt{\rho} \sin\left(\frac{\theta}{2}\right) \sin(\theta), \sqrt{\rho} \cos\left(\frac{\theta}{2}\right) \sin(\theta) \end{array} \right\}^T$$



**Remark:** this is the same approximation space as used in the XFEM (Moës, Dolbow, Belytschko) but in a BEM sense.

# Results – mode I problem

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Introduction

Reciprocal theorem

Fundamental solutions

Boundary Integral Equation

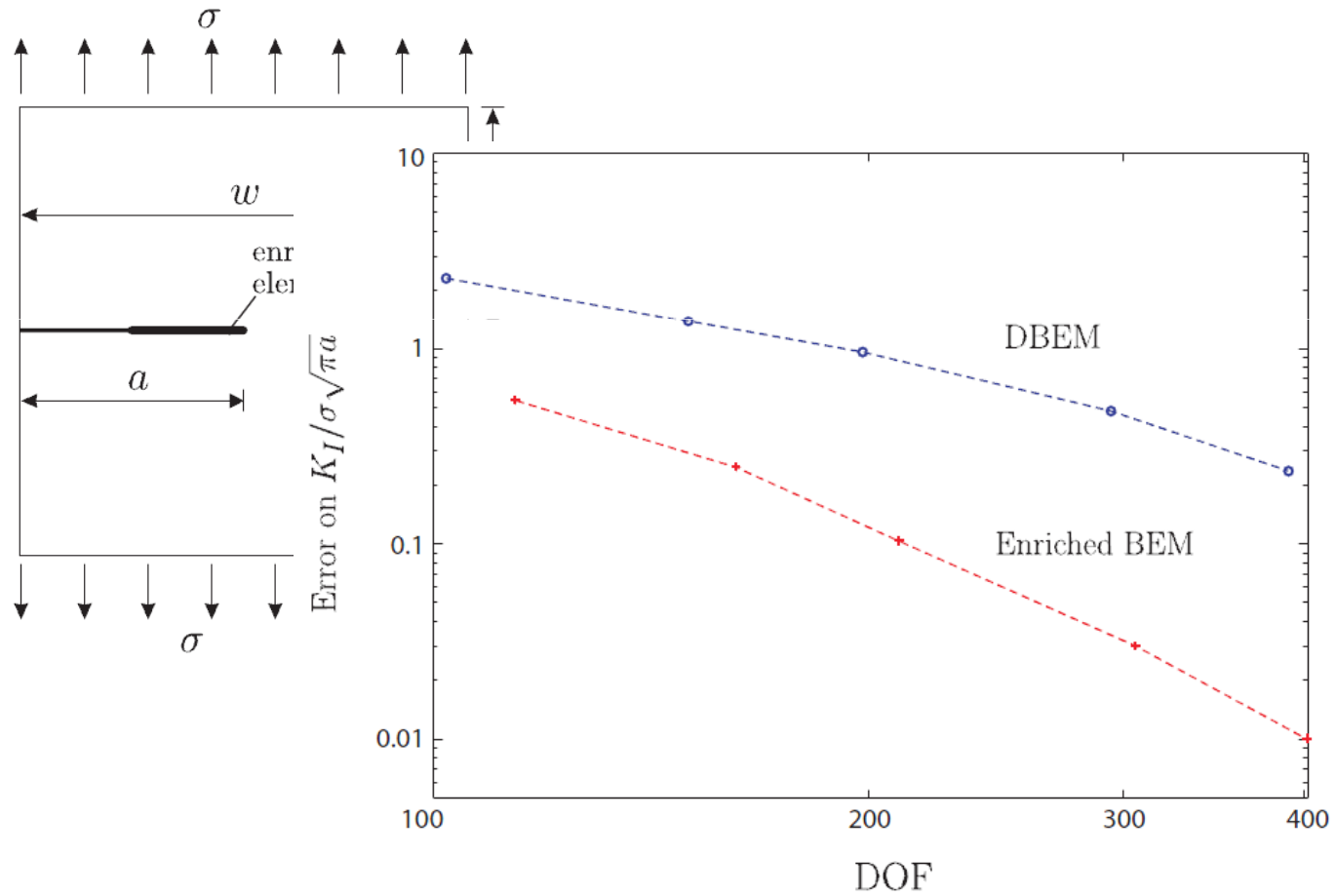
Boundary Element Method

Re-analysis and Interactivity

Demonstration

Enrichment

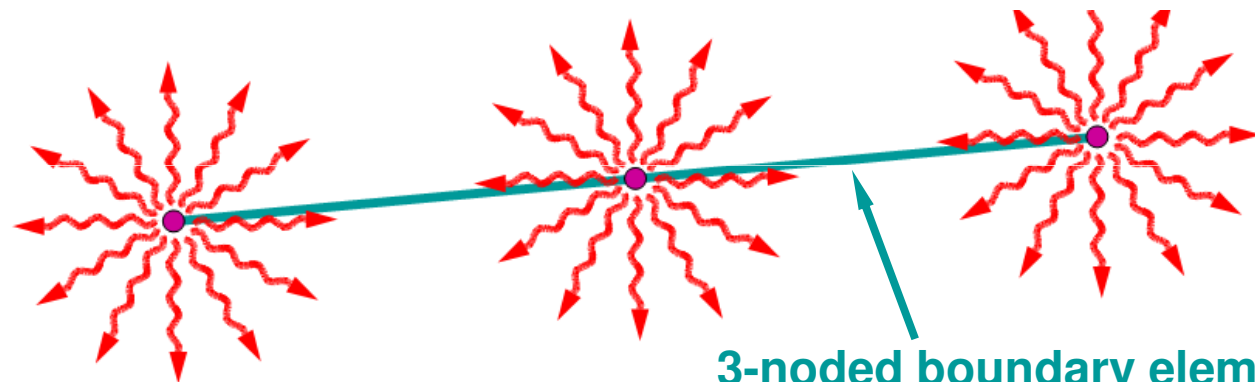
## Edge crack in finite plate



## PU-BEM enriched basis for wave problems

The PUM multiple plane wave expansion for potential on an element

$$\phi(x) = \sum_{j=1}^J N_j(x) \sum_{m=1}^M A_{jm}^e e^{ik\psi_{jm}^e \cdot x}, \quad |\psi_{jm}^e| = 1$$

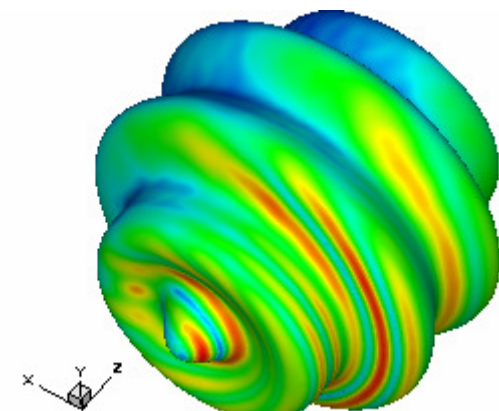
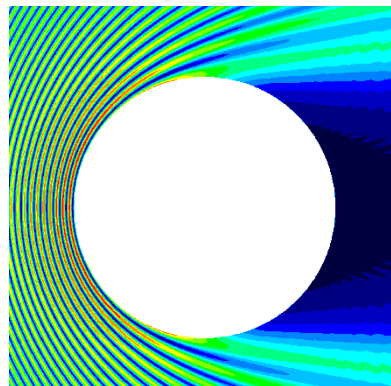


**12 waves at the node**

$M = 12$

**3-noded boundary element**

$J = 3$



*ACME School*  
*4<sup>th</sup> April 2011*

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## Conclusions

- BEM has been presented for elastostatics problems
- Body forces and non-linearity can be handled by further treatment not discussed here
- Attractive for various classes of problem
- Re-analysis leads to interactivity in stress analysis
- Enrichment of the approximation space can yield improved accuracy